EECE 360 Lecture 28

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State Feedback

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Chapter 11.1-11.6, 11.9, 11.10

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Review: Controllability

- The eigenvalues of (A-BK) can be arbitrarily assigned when the system [A,B,C,D] is controllable.
- A system is **controllable** if there exists a control u(t) that can transfer any initial state x(0) to any desired state x(t) in a finite time T.
- The controllability matrix

$$S_C = [B \ AB \ A^2B \ \cdots \ A^{n-1}B]$$

must have rank *n* for the system [A,B,C,D] to be controllable. (S_C is "full-rank".)

• When S_C is full-rank, det $(S_C) \neq 0$

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- Review
 - Controllability test
 - Ackermann's formula for controller design
- Today
 - Observability
 - Observerability test
 - Observer design through Ackermann's
 - Separation principle
 - Combining observers and controllers

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Review: Ackermann's Formula

The state feedback gain matrix

 $K = [k_1 \quad k_2 \quad \cdots \quad k_n]$ where u(t) = r(t) - Kx(t) that produces the desired characteristic equation

is given by
$$q(s) = s^n + \alpha_1 s^{n-1} + \dots + \alpha_n$$

where

$$K = \begin{bmatrix} 0 & 0 & \cdots & 1 \end{bmatrix} S_{\mathsf{C}}^{-1} q(A)$$

$$S_{\mathsf{C}} = [B \quad AB \quad \cdots \quad A^{n-1}B] \text{ and } q(A) = A^n + \alpha_1 A^{n-1} + \cdots + \alpha_n I$$

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Output feedback

- Often, it is not feasible or even possible to measure all components of the state directly
- The output encapsulates a subset of the states which can be measured.
- For example, in the spring-mass-damper system, only the position of the mass is measured
- In this case, the remaining states must be accurately estimated using an observer



Output feedback

Since the control law acts upon the estimated value of the state

 $u = -K\hat{x}$

 The observer must be designed such that the estimate of the state is guaranteed to converge to the actual value of the state

 $e=x-\hat{x}$

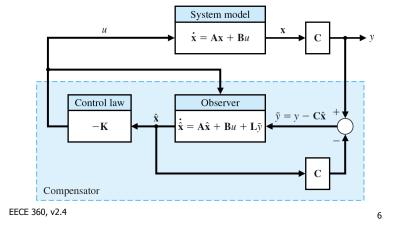
 The estimate is a dynamic process which evolves over time according to

$$\dot{e} = \dot{x} - \dot{\hat{x}}$$



Output feedback

Output-based regulation





Output feedback

• We know that $\dot{x} = Ax + Bu$

y = Cx

And so we create an estimated system

 $\dot{\hat{x}} = A\hat{x} + Bu + L(y - C\hat{x})$

- Which is dependent on the difference between the actual output and the output value expected based on the current estimate of the state
- Therefore the error *e* evolves according to

$$\dot{e} = \dot{x} - \dot{\hat{x}} = Ax - A\hat{x} + L(Cx - C\hat{x})$$
$$= (A - LC)e$$

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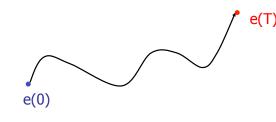
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- The eigenvalues of (A-LC) can be arbitrarily assigned when the system is **observable**.
- A system is **observable** if there exists a finite time T such that, given the input u(t), the initial state x(0) can be determined from the observation history y(t).



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Observability vs. Controllability

- Note that the observer gain *L* is a matrix of dimension *n* x *p*, where the output matrix *C* is *p* x *n*
- For a SISO system, *L* is *n* x 1
- Therefore *LC* will be an *n* x *n* matrix that can be subtracted, element-wise, from *A*.
- By contrast, recall that the controller gain K is a matrix of dimension m x n, where the input matrix B is m x n
- For a SISO system, K is 1 x n
- Therefore *BK* will be an *n* x *n* matrix that can be subtracted, element-wise, from *A*.



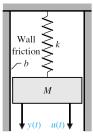
Observability



Example: Spring-Mass-Damper

System and input matrices

$$A = \begin{bmatrix} 0 & 1 \\ -\frac{k}{M} & -\frac{b}{M} \end{bmatrix}, \quad C = \begin{bmatrix} 1 & 0 \end{bmatrix}$$

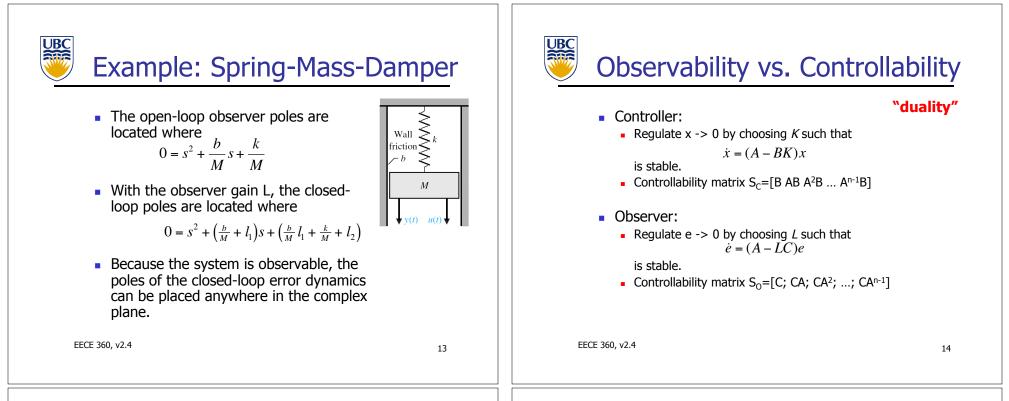


Observability matrix

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$$S_O = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

- To test for controllability, $|S_0|=1-0=1$
- Therefore the system is **observable**.





Observability vs. Controllability

- Controller:
 - Design a control gain K =[k₁ k₂ k₃ ... k_n] through Ackermann's formula

$$K = \begin{bmatrix} 0 & \dots & 0 & 1 \end{bmatrix} S_C^{-1} q(A)$$

- Observer:
 - Design an observer gain $L = [I_1 \ I_2 \ I_3 \ \dots \ I_n]^T$ through Ackermann's formula

$$L = q(A)S_0^{-1}[0 \dots 0 1]^T$$

 This takes advantage of the **duality** between the observer and controller

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Example: Consider the spring-mass-damper system

- Choose the closed-loop poles of the observer to be 4-10 times faster than the controller poles
- For now, assume that these poles occur at a desired damping ς and desired natural frequency ω_n , the characteristic equation is

$$q(s) = s^2 + 2\overline{\zeta}\overline{\omega}_n s + \overline{\omega}_n^2$$

Compute the observability matrix and its inverse

$$S_{O} = \begin{bmatrix} C \\ CA \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$
$$S_{O}^{-1} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

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• The characteristic equation in terms of A is $q(A) = A^{2} + 2\overline{\xi}\overline{\omega}_{n}A + \overline{\omega}_{n}^{2}, \text{ therefore the control gain is}$ $L = \left(A^{2} + 2\overline{\xi}\overline{\omega}_{n}A + \overline{\omega}_{n}^{2}I\right) \begin{bmatrix}0 & 1\\1 & 0\end{bmatrix} \begin{bmatrix}0 & 1\end{bmatrix}^{T}$ $= \left(\begin{bmatrix}0 & 1\\-\frac{k}{M} & -\frac{b}{M}\end{bmatrix}^{2} + 2\overline{\xi}\overline{\omega}_{n}\begin{bmatrix}0 & 1\\-\frac{k}{M} & -\frac{b}{M}\end{bmatrix} + \overline{\omega}_{n}^{2}I\right) \begin{bmatrix}0 & 1\\1 & 0\end{bmatrix} \begin{bmatrix}0\\1\end{bmatrix}$ $= \left(\begin{bmatrix}-\frac{k}{M} & -\frac{b}{M}\\\frac{kb}{M^{2}} & -\frac{k}{M} + \frac{b^{2}}{M^{2}}\end{bmatrix} + 2\overline{\xi}\overline{\omega}_{n}\begin{bmatrix}0 & 1\\-\frac{k}{M} & -\frac{b}{M}\end{bmatrix} + \overline{\omega}_{n}^{2} \begin{bmatrix}1 & 0\\0 & 1\end{bmatrix}\right) \begin{bmatrix}0\\1\end{bmatrix}$ $= \left(\begin{bmatrix}-\frac{b}{M}\\-\frac{k}{M} + \frac{b^{2}}{M^{2}}\end{bmatrix} + 2\overline{\xi}\overline{\omega}_{n}\begin{bmatrix}1\\-\frac{b}{M}\end{bmatrix} + \overline{\omega}_{n}^{2}\begin{bmatrix}0\\1\end{bmatrix}$ EECE 360, v2.4



Ackermann's Formula

 The observer gain to achieved the desired closedloop poles for the error dynamics is

$$L = \begin{bmatrix} -\frac{b}{M} \\ \frac{k}{M} + \frac{b^2}{M^2} \end{bmatrix} + 2\overline{\xi}\overline{\omega}_n \begin{bmatrix} 1 \\ -\frac{b}{M} \end{bmatrix} + \overline{\omega}_n^2 \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$
$$L = \begin{bmatrix} 2\overline{\xi}\overline{\omega}_n - \frac{b}{M} \\ \overline{\omega}_n^2 - 2\overline{\xi}\overline{\omega}_n \frac{b}{M} - \frac{k}{M} + \frac{b^2}{M^2} \end{bmatrix}$$

 Note that the observer gain will drive the error dynamics to the desired closed-loop error dynamics poles.

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Ackermann's Formula

• The closed-loop system is $\dot{e} = (A - LC)e$ $= \left(\begin{bmatrix} 0 & 1 \\ -\frac{k}{M} & -\frac{b}{M} \end{bmatrix} - \begin{bmatrix} 2\overline{\xi}\overline{\omega}_n - \frac{b}{M} \\ \overline{\omega}_n^2 - 2\overline{\xi}\overline{\omega}_n - \frac{b}{M} - \frac{k}{M} + \frac{b^2}{M^2} \end{bmatrix} \begin{bmatrix} 1 & 0 \end{bmatrix} \right) x$ $= \left(\begin{bmatrix} 0 & 1 \\ -\frac{k}{M} & -\frac{b}{M} \end{bmatrix} - \begin{bmatrix} 2\overline{\xi}\overline{\omega}_n - \frac{b}{M} & 0 \\ \overline{\omega}_n^2 - 2\overline{\xi}\overline{\omega}_n - \frac{b}{M} - \frac{k}{M^2} + \frac{b^2}{M^2} & 0 \end{bmatrix} \right) x$ $= \left[\frac{-2\overline{\xi}\overline{\omega}_n + \frac{b}{M}}{\overline{\omega}_n^2 + (2\overline{\xi}\overline{\omega}_n - \frac{b}{M}) \frac{b}{M}} - \frac{b}{M} \right] x$ • which has poles at $0 = |s - (A - LC)| = s^2 + 2\underline{\zeta}\underline{\omega}_n s + \underline{\omega}_n^2$ EECE 360, v2.4



Using Matlab

• Designing controller gains • $K = [0 \ \dots \ 0 \ 1]S_{C}^{-1}q(A)$ • K = acker(A,B,Pk)Use 'place' for MMO systems • $L = q(A)S_{O}^{-1}[0 \ \dots \ 0 \ 1]^{T}$ • LT = acker(A',C',Pl)• L = LT'• K = hat the the transpose of both A and C required!

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Controllability Summary

- A system (A,B,C,D) is controllable if its controllability matrix S_c is full rank.
- The closed-loop poles of a controllable system can be placed anywhere in the complex plane.
- Choose the desired pole location, then compute the gain K required to achieve those locations
- Ackermann's formula for SISO systems (Matlab's `acker')
- Matlab's `place' for MIMO systems

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Observability Summary

- A system (A,B,C,D) is observable if its observability matrix S₀ is full rank.
- The closed-loop poles of the error dynamics of an observable system can be placed anywhere in the complex plane.
- This allows arbitrarily fast convergence of the state estimate to the actual value of the state.
- Choose the desired error pole location, then compute the gain L required to achieve those locations
- Ackermann's formula for SISO systems (Matlab's 'acker') with transposed matrices

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Observers/controllers

- The dynamics for *dx/dt* and *de/dt* are coupled
 - State dynamics $\dot{x} = Ax + Bu, \quad u = -K(x + e)$
 - = (A BK)x + BKe
 - Error dynamics

$$\dot{e} = \dot{x} - \dot{\hat{x}}, \quad \dot{\hat{x}} = A\hat{x} + Bu + L(y - C\hat{x})$$
$$= Ax + Bu - A(x + e) - Bu - LCx + LC(x + e)$$
$$= (A - LC)e$$

Observers/controllers

In state-space form, with

$$\tilde{x} = \begin{bmatrix} x \\ e \end{bmatrix}$$

 The closed-loop system and observer dynamics are

$$\begin{bmatrix} \dot{x} \\ \dot{e} \end{bmatrix} = \begin{bmatrix} A - BK & BK \\ 0 & A - LC \end{bmatrix} \begin{bmatrix} x \\ e \end{bmatrix}$$

 The eigenvalues of this system are eig(A-BK) and eig(A-LC)



Separation Principle **

- Although the state dynamics and observer dynamics are coupled, the controller and the observer can be designed independently
- Standard procedure:
 - Design a controller with gain K to place the roots of (A-BK) at desired locations in the LHP.
 - Design an observer with gain L to place the roots of (A-LC) at desired locations in the LHP.
- Generally the observer poles are placed such that the observer dynamics are 4-10 times faster than the state dynamics.

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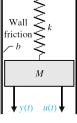


Example: Spring-Mass-Damper

 The open-loop system poles are located where

$$0 = s^2 + \frac{b}{M}s + \frac{k}{M}$$

• With controller gain *K* and observer gain L, the closed-loop poles of the extended system are located where

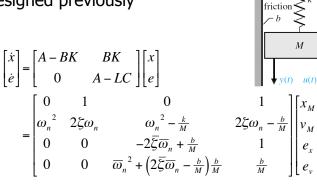


$$0 = \left(s^{2} + 2\zeta\omega_{n}s + \omega_{n}^{2}\right)\left(s^{2} + 2\overline{\zeta}\overline{\omega}_{n}s + \overline{\omega}_{n}^{2}\right)$$



Example: Spring-Mass-Damper

Using the controller and observer designed previously



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Example: Spring-Mass-Damper

- Because the system is controllable and observable, the closed-loop poles of the error dynamics and the system dynamics can be placed arbitrarily.
- Wall friction \swarrow^{b} М

Wall

М

- However, the further away the closedloop poles are placed from the openloop poles, the higher the control effort.
- Additionally, excessively high observer gains can lead to amplification of noise inherent to the output measurements.



- Controllability matrix S_c to test whether it is possible to put the poles of the closed-loop state dynamics in any desired location
- Observability matrix S₀ to test whether it is possible to put the poles of the closed-loop error dynamics in any desired location
- Duality of controller (with gain K) and observer (with gain L)
- **Separation principle** allows independent design of the controller and observer

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